

Total Lagrangian SPH Modelling of Necking and Fracture in Electromagnetically Driven Rings

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Received: date / Accepted: date

Abstract This paper describes research on the prediction of necking and failure in metals at very high strain rates. The model developed in this paper uses a total Lagrangian SPH formulation with a normalised kernel. The detailed data from electromagnetically driven ring experiments by Zhang and Ravi-Chandar (2006) is used to evaluate the accuracy of the model predictions. In order to correctly model fracture in the total Lagrangian SPH formulation a visibility criterion based on a truncated cone has been implemented to remove particles obscured by a failed particle. A Johnson-Cook plasticity model is used in combination with a Lemaitre damage model to describe the plastic deformation and fracture of the rings. The effect of Joule heating due to the current induced in the ring is taken into account in the constitutive model. The acceleration due to the ring currents was implemented in the SPH code as a body force. The results demonstrate that this type of model is capable of predicting the number of fragments as well as the time of fracture. In agreement with experimental data, the model also predicts arrested necks and bending in the fragments.

Keywords smoothed particle hydrodynamics · meshless · modelling · fracture · expanding ring · necking · strain-rate

1 Introduction

Due to their potential in dealing with fracture, meshless methods, such as Smoothed Particle Hydrodynamics (SPH), are of particular interest for the prediction of fragmentation and fracture at high strain rates. This paper describes research on the numerical prediction of necking and fracture of electromagnetically driven rings using the SPH method. In an electromagnetically driven ring experiment a metal ring is radially accelerated by an electromagnetic force acting on the ring. These electromagnetic forces are generated by inducing a current in the ring by discharging a capacitor into a coil located inside the ring. A schematic representation of this setup is shown in Figure 1. There are a number of publications with detailed experimental results as well as numerical models of electromagnetically driven expanding ring fragmentation. In the following paragraphs a brief overview is provided of the most relevant papers.

1.1 Electromagnetically Driven Ring Experiments

A widely cited paper on electromagnetically driven expanding ring experiments is by Grady and Benson (1983). This paper describes an electromagnetical expanding ring test method in detail and presents results for copper and aluminium rings. They show the influence of ring velocity on fracture strain and the dependence of fragment number on kinetic energy. The data in this paper is frequently used to validate numerical models of

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ring fragmentation. Satapathy and Landen (2006) also describe an electromagnetically driven expanding ring experiment. They study the effect of adiabatic heating, due to the current induced in the rings, on the mechanical properties during expansion, and compare this with equivalent isothermal properties. The paper does not discuss fragmentation of the rings. In a second paper (Landen et al., 2009) they extend their experiments to discuss the effect of (rapid) preheating on the mechanical properties and their effect on expansion of the rings. A good review of fragmentation models and experiments can be found in the paper by Zhang and Ravi-Chandar (2009c). One of the conclusions in this paper is that for fragmentation to be captured appropriately models must account both for damage localization and for statistical variations in this localization. The series of articles by Zhang and Ravi-Chandar (Zhang and Ravi-Chandar, 2006, 2008, 2009a,b) provides a large amount of data on ring fragmentation experiments. In Zhang and Ravi-Chandar (2006) they test Al 6061-0 specimens. The rings have a thickness of 0.5mm, length of 1.0mm and a mean radius of 15.5mm. Expansion speeds were in a range of 80 to 200m/s. Their analysis of neck development and fragment distribution lead them to conclude that the spacing between the necks and fragment length is determined by the presence of material imperfections and the propagation of Mott release waves. Based on the measured currents in the ring they calculate the temperature increase in the ring. Depending on the expansion speed the ring reached temperatures of 300-400°C. This means that it is important to take into account the effect of temperature on the mechanical properties in numerical modelling of the process. It is shown (Zhang and Ravi-Chandar, 2006) that the average strain in the fragments stays below the Considère strain, and that the apparent increase of ductility at high strain rates, reported in other papers, is due to the measurement method which includes the higher strains in arrested necks within the fragments in the average strain of the fragment. The strain in the fragments (excluding necks) they measured were of the order of 0.22, which is very close the static failure strain of 0.25. The speed at which a localised deformation/damage spreads across the whole ring or cylinder section (and hence develops in a fracture) is identified as important (Zhang and Ravi-Chandar, 2006, 2009b). Therefore an appropriate 3D model must have sufficient mesh resolution across a section of the ring. If the mesh resolution is insufficient, or a 1D analysis is used localisation extends across the whole section of the ring too fast (in the case of single element cross section instantaneously). In Zhang and Ravi-Chandar (2009b) they stated that for quantitative predictions of

the fragment distribution, it is necessary to develop better models of the process of ductile failure that occurs within the neck (or shear localized) regions.

1.2 Electromagnetically Driven Ring Models

Several models of the electromagnetically driven ring experiment have been published. Most models use Lagrangian FE codes (Sørensen and Freund, 2000; Pandolfi et al., 1999; Becker, 2002; Rusinek and Zaera, 2007; Zhang and Ravi-Chandar, 2008, 2009a), although sometimes Eulerian hydrocodes (Meulbroek et al., 2008) or finite difference models (Hu and Daehn, 1996; Johnson, 1981) are also used. The fragmentation stage is not always modelled with models restricting themselves to predicting the expansion speed and strain rate. A number of one dimensional models has been developed. Johnson (1981) proposed a 1D model for expanding ring fragmentation based on void growth and used an initial random distribution of porosity (imperfection). This 1D model was solved for a known strain rate history using the finite difference method. Hu and Daehn (1996) also presented a 1D finite difference model for dynamic tensile and expanding ring tests. They used a Holomon-type material law with power law strain rate sensitivity. They analysed the effect of velocity, and parameters of the constitutive law (strain hardening exponent and strain rate sensitivity exponent) on ductility in tensile and expanding ring tests. The results compare well with experiment data from the literature. Zhou, Molinari and Ramesh use cohesive elements in a 1D model to analyse the fragment size distribution in radially expanding rings of brittle material (Zhou et al., 2006a) and elasto-visco-plastic material (Zhou et al., 2006b). Mercier and Molinari (2004) developed a linear perturbation analysis for the radial expansion of rings. They studied the effect of strain rate sensitivity and strain hardening on fragment size. Fyfe and Rajendran (1979) used perturbation analysis to study the effect of rate sensitivity and inertia effects on the strain at onset of necking. Triantafyllidis and Waldenmyer (2004) studied the effect of variations in strain hardening, strain rate sensitivity and temperature sensitivity on onset of necking strain using an analytical model. The model considered the coupled electromagnetic and thermomechanical problem. Shenoy and Freund (1999) studied the effect of expansion speed on the number of necks and they also concluded from their perturbation analysis that the number of necks is independent of the strain hardening exponent. Guduru and Freund (2002) analyse the effect of expansion speed on the number of necks and bifurcation strain. More recently a linear perturbation analysis of necking in hemispherical shells

(Mercier et al., 2010) has been performed. This work also showed the influence of the constitutive law on the predicted number of necks. Vadillo et al. (2012) performed a linear perturbation analysis to study the effect of strain rate and strain rate sensitivity on localisation in expanding rings.

Sørensen and Freund (2000) presented a 2D finite element model of an expanding ring. The study focussed on the development of necking, and fragmentation of ductile materials. They used a Gurson model to represent void growth. The rings were loaded by applying a sinusoidal body force pulse to represent the electromagnetic force acting on the ring. Imperfections in their model were introduced as variations in ring thickness. The necking was shown to be quite sensitive to the perturbation of the ring's thickness. This led to the conclusion that this type of perturbation is not appropriate for the prediction of fragment number and size. Becker (2002) presented 3D finite element modelling results of electromagnetically driven expanding ring experiments. The loading was applied by prescribing the radial velocity of the ring. Damage was modelled with a Gurson model. Material failure was determined with the Drucker stability criterion and a bifurcation model. Realistic strain to fracture and number of fragments were obtained, but no comparison with experimental data was made. Pandolfi et al. (1999) simulated the expanding ring experiments presented by Grady and Benson (1983). The simulation model in their paper used cohesive elements to model fracture. They demonstrated the ability of their method to model fragment number and size by comparison to the experiments of Grady and Benson (1983). However in their model only one element through the thickness of the ring was used. Consequently, as an element fails it immediately generates a fully fractured ring and release waves start propagating from the free surface. The same mesh approach is used by Rusinek and Zaera (2007). They use the Rusinek-Klepaczko constitutive equation and focus on the effect of the strain hardening, thermal softening and strain rate sensitivity terms in the constitutive equation, and the effect of Critical Impact Velocity on fragment size. Meulbroek et al. (2008) used CTH to simulate expanding ring fragmentation of Zhang and Ravi-Chandar (2006). Their model combined a 2D rectangular mesh with Bammann-Chiesa-Johnson constitutive law and Mie-Grüneisen equation of state. This constitutive model includes a void growth damage model. Due to limitations of the CTH software the loading of the ring was modelled by applying an initial velocity. For the same reason no random distribution of initial defects was applied. The model had problems with convergence and voids developed unrealistically early on

in the simulation. The authors of this paper concluded that in order to predict the correct average fragment size and distribution, and the correct failure mechanism it was necessary to accurately model the necking mechanism (for the rings). Also the convergence issue they identified with the use of Eulerian codes like CTH needs to be resolved. Randles and Libersky (1996) were the first to apply the conventional SPH method to simulate fragmentation of explosively driven rings and cylinders. Their results show a very good correlation in terms of fragment mass statistics. Apart from Randles and Libersky's work, Hiroe et al. (2008) used 2D SPH in combination with Mott's stochastic failure model to assess 22 different explosively driven cylinder test configurations. Rabczuk and Belytschko (2004b) presented 2D cylinder fragmentation using the Element Free Galerkin Method (EFGM). In a recent paper by Dong et al. (2008) the Reproducing Kernel Particle Method (RKPM) was used to study failure of pressurised cylinders with an initial crack present. The pressure loading in this model was either applied as a step function at the beginning of the simulation or increased linearly over time. Note that this type of loading is different from the impulsive loading in explosively or electromagnetically driven cylinders. The work presented in this paper is an attempt to demonstrate that the SPH method in combination with a suitable material model is capable of quantitatively predicting fragmentation (fragment distributions and fracture time) in expanding ring experiments. The expanding ring experiments by Zhang and Ravi-Chandar (Zhang and Ravi-Chandar, 2006) were used to validate the model accuracy.

2 Description of Electromagnetically Driven Ring Experiments

As already stated, the experimental data in Zhang and Ravi-Chandar (2006) was used to evaluate the accuracy of the model predictions. In these experiments electromagnetic induction was used to accelerate aluminium 6061-0 rings. A high speed camera with combined framing and streak imaging capability was used to provide detailed information about the necking and fracture of these rings. The experimental setup used is shown in Figure 1. In this figure the voltage source V is in reality a high voltage condenser with high capacitance (of the order of μF). When the switch is closed a current I_1 runs through the coil, around which the ring is placed. The resistance of this primary circuit is R_1 and the inductance of the coil is L_{11} . The current pulse induces a current I_2 in the ring (which has resistance R_2 and L_{22}). The mutual inductance between the two coils is

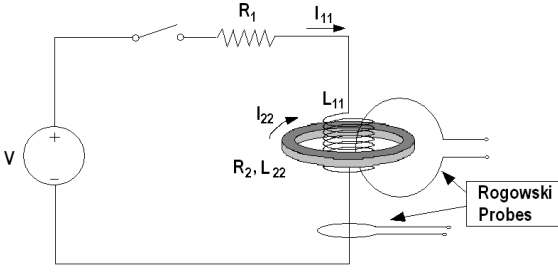


Fig. 1 Experimental Setup used by Zhang and Ravi-Chandar (Zhang and Ravi-Chandar, 2006)

L_{21} . Both currents I_1 and I_2 are measured by Rogowski probes. The rings had a thickness of 0.5mm, length of 1.0mm and a mean radius of 15.5mm. The high voltage condensor is charged with voltages of 5, 6, 7 and 8kV. This resulted in radial expansion speeds ranging from 80 to 200m/s.

3 Conventional SPH

The Smoothed Particle Hydrodynamics (SPH) method is based on the convolution principle. The approximation $\langle F \rangle$ or smoothed value of a field F at a point x_i is given by:

$$\langle F(x_i) \rangle = \int F(x) W(|x_i - x|, h) dV \quad (1)$$

Where $W(|x_i - x|, h)$ is the kernel function and h is the smoothing length. This kernel function has to have compact support and possess the following properties:

$$\int W(|x_i - x|, h) dV = 1 \quad (2)$$

$$W(|x_i - x|, h) \xrightarrow{h \rightarrow 0} \delta \quad (3)$$

If the domain is discretised with a number of particles then the convolution integral is approximated by a sum as follows:

$$\langle F(x_i) \rangle \approx \sum F(x_j) W(|x_i - x_j|, h) \frac{m_j}{\rho_j} \quad (4)$$

Gradients of functions are similarly calculated:

$$\langle \nabla F(x_i) \rangle \approx \sum F(x_j) \nabla W(|x_i - x_j|, h) \frac{m_j}{\rho_j} \quad (5)$$

Applied to hydrocodes these approximations are used to discretise the conservation laws. In the spatial description the conservation of mass, momentum and energy laws are as follows:

$$\dot{\rho} = -\rho \nabla \cdot v \quad (6)$$

$$\rho \ddot{u} = \nabla \cdot \sigma + b \quad (7)$$

$$\rho \dot{e} = -\sigma : \dot{\epsilon} \quad (8)$$

which results in (Vignjevic et al., 2009):

$$\langle \dot{\rho}_i \rangle = -\rho_i \sum_{j=1}^{np} \frac{m_j}{\rho_j} (v_i - v_j) \cdot \nabla_{x_i} W(|x_i - x_j|, h) \quad (9)$$

$$\langle \ddot{u}_i \rangle = \sum_{j=1}^{np} \frac{m_j}{\rho_j} (v_j - v_i) (v_R \cdot \nabla_{x_i} W(|x_i - x_j|, h)) - \sum_{j=1}^{np} m_j \left(\frac{\sigma_i}{\rho_i^2} + \frac{\sigma_j}{\rho_j^2} \right) \nabla_{x_i} W(|x_i - x_j|, h) + b \quad (10)$$

$$\langle \dot{e}_i \rangle = \frac{\sigma_i}{\rho_i} : \sum_{j=1}^{np} \frac{m_j}{\rho_j} (v_i - v_j) \nabla_{x_i} W(|x_i - x_j|, h) \quad (11)$$

Within the context of a transient dynamics code, the strain rates are required for the constitutive update. They are obtained using the following calculation of the velocity gradient:

$$\langle \nabla v_i \rangle = \sum_{j=1}^{np} \frac{m_j}{\rho_j} (v_j - v_i) \nabla W(|x_i - x_j|, h) \quad (12)$$

It can be seen that the kernel functions which appear in the discretised conservation equations in spatial description are evaluated using the current positions of particles. The kernels evaluated using current particle positions are called Eulerian kernels (Vignjevic et al., 2006; Rabczuk and Belytschko, 2004a).

The equations above are symmetrised (ie. they use differences or sums of variables at particles i and j). This symmetrisation has the advantage of preserving local conservation of momentum and energy.

Although frequently used the SPH method formulated in this way suffers from tensile instability. The tensile instability was first identified by Swegle et al. (1995) and manifests itself as the development of unphysical fractures or clumping together of particles when the material is subjected to tension. Belytschko and Xiao (2000), Rabczuk and Belytschko (2004a) and Vignjevic et al. (2006) showed that the tensile instability associated with the use of an Eulerian kernel can be avoided by using a Total Lagrangian kernel. Removing tensile instability is crucial in modelling the expanding ring problem as the rings are subjected to large magnitude tensile stresses during the homogeneous expansion phase. The SPH code developed in the Applied Mechanics Department at Cranfield University was used for all simulations. The implementation of the Total Lagrangian formulation in this code is described in (Vignjevic et al., 2006) and will be summarised in the next section.

4 Total Lagrangian SPH

The Total Lagrangian kernel function is evaluated in the initial configuration, in other words the initial coordinates x_i^0 and x_j^0 are used to evaluate the kernel W for given i and j particles.

In the Total Lagrangian SPH formulation the balance equations are written in the initial configuration (material description):

$$\rho J = \rho^0 \quad (13)$$

$$\rho^0 \ddot{u} = \nabla_0 \cdot P + b \quad (14)$$

$$\rho^0 \dot{e} = P : \dot{F} \quad (15)$$

where $J = |F|$ is the Jacobian of the deformation gradient, ρ density, u displacement, P first Piola Kirchhoff stress, b body force per unit mass, e internal energy. The superscript 0 indicates initial configuration.

The SPH discretisation (Vignjevic et al., 2009, 2006) results in the following equation for the deformation gradient:

$$\langle F_i \rangle = - \sum_{j=1}^{np} \frac{m_j}{\rho_j^0} (v_i - v_j) \nabla_{x_i^0} W(|x_i^0 - x_j^0|, h^0) \quad (16)$$

From this equation the conservation of mass can be used to calculate the current density:

$$\rho = J^{-1} \rho^0 \quad (17)$$

The conservation of momentum equation becomes:

$$\langle \ddot{u}_i \rangle = - \sum_{j=1}^{np} m_j \left(\frac{P_i}{\rho_i^{02}} + \frac{P_j}{\rho_j^{02}} \right) \nabla_{x_i^0} W(|x_i^0 - x_j^0|, h^0) + b \quad (18)$$

The energy update is calculated as follows:

$$\langle \dot{e}_i \rangle = \frac{P_i}{\rho_i} : \sum_{j=1}^{np} \frac{m_j}{\rho_j^0} (v_i - v_j) \nabla_{x_i^0} W(|x_i^0 - x_j^0|, h^0) \quad (19)$$

The constitutive update in Cranfield University's SPH code uses a rate of deformation tensor $\dot{\epsilon}$ to calculate a stress increment. The rate of deformation tensor can be obtained from the deformation gradient using the Green-Lagrange strain rate tensor $\dot{E}$:

$$\dot{E} = \frac{1}{2} (\dot{F}^T F + F^T \dot{F}) \quad (20)$$

and the following expression:

$$\dot{\epsilon} = F^{-T} \dot{E} F^{-1} \quad (21)$$

The Cauchy stress σ obtained from the constitutive update is transformed to the 1st Piola-Kirchhoff stress P for use in the conservation of momentum equation using:

$$P = J F^{-1} \sigma \quad (22)$$

A cubic B-spline kernel function was used. The kernel functions were normalised (Shepard functions) to restore zero order consistency:

$$W(|x_i^0 - x_j^0|, h^0) = \frac{\tilde{W}(|x_i^0 - x_j^0|, h^0)}{\sum_{j=1}^{np} \frac{m_j}{\rho_j^0} \tilde{W}(|x_i^0 - x_j^0|, h^0)} \quad (23)$$

Where $\tilde{W}$ is the standard cubic B-spline kernel. These semi-discretised equations were integrated in time using a central difference integration scheme (explicit time integration). The update of particle positions was performed using a smoothed velocity (XSPH) (Randles and Libersky, 1996). The motivation for using the SPH method is that due to its meshless nature the SPH method is less directionally sensitive compared to a finite element mesh which has preferential directions for cracks growth or localisation. Furthermore the smoothing properties of the kernel interpolation in the SPH method may make it less sensitive to deformation localisation related to material softening.

4.1 Constitutive Behaviour

The constitutive model is based on an additive decomposition of the strain rate tensor. Elastic behaviour was assumed to be isotropic and is defined through the generalised Hooke's law. Plasticity is described using a J2 flow theory with isotropic hardening. The modified Johnson Cook yield function and an associative flow rule are used. The Jaumann stress rate is used to update the stress tensor and a radial return algorithm is used for the integration of the constitutive equations. The volumetric part of deformation is described by the Mie-Gruneisen equation of state. The Johnson-Cook material model is routinely used for problems where high strain rates and temperatures occur. Similarly to Randles and Libersky (1996) a modification to the Johnson-Cook model was made because the original Johnson-Cook model is limited to strain rates up to 10^3 s^{-1} . The modification used in this work is by adding an extra term:

$$\left(\frac{\dot{\epsilon}}{d} \right)^E \quad (24)$$

This term is identical to the term of the Cowper-Symonds strain rate dependent model as shown in Equation 25.

$$\sigma_Y = \left(A + B \bar{\epsilon}_{pl}^n \right) \left(1 + C \ln \left(\frac{\dot{\epsilon}}{\dot{\epsilon}_0} \right) + \left(\frac{\dot{\epsilon}}{d} \right)^E \right) (1 - T^{*m}) \quad (25)$$

In addition to the Johnson-Cook model the Mie-Gruneisen equation of state was used (LSTC, 2007).

Table 1 Parameters for Johnson-Cook model used for aluminium 6061-O

Property	Value
G	77.0GPa
A	25.0MPa
B	185MPa
n	0.28
C	0.002
d	0.08
E	0.5
m	0.75
T_m	925.0K
T_r	293.0K
$\dot{\epsilon}_0$	1.0s ⁻¹
Γ	1.97
c	5240m/s
S_1	1.4
S_2	0.0

The values for Γ , c , S_1 , S_2 are those for aluminium 6061-T6 and were taken from Corbett (2006) and summarised in Table 1. The values for the other material model parameters were mainly based on the data provided in Zhang and Ravi-Chandar (2006). The data is summarised in Table 1. The value of c is taken from Corbett (2006). G is the shear modulus of the material.

4.2 Damage Model

A combination of the Johnson-Cook failure model with a ductile damage model developed by Lemaitre was used in this work. The phenomenological Lemaitre (Lemaitre and Sermage, 1997) damage model is based on thermodynamics of irreversible processes. It can be used to model brittle and ductile isotropic damage. For ductile damage strain to fracture strongly decreases when triaxiality ratio increases. The Lemaitre damage model describes the damage growth rate $\dot{D}$ as a function of the effective plastic strain rate $\dot{\epsilon}_{pl}$ as follows (more details can be found in Lemaitre and Chaboche (2007); Lemaitre and Sermage (1997)):

$$\dot{D} = \begin{cases} \left(-\frac{Y}{S}\right)^t \dot{\epsilon}_{pl} & \text{if } \bar{\epsilon}_{pl} \geq \bar{\epsilon}_{threshold} \\ 0 & \text{if } \bar{\epsilon}_{pl} < \bar{\epsilon}_{threshold} \end{cases} \quad (26)$$

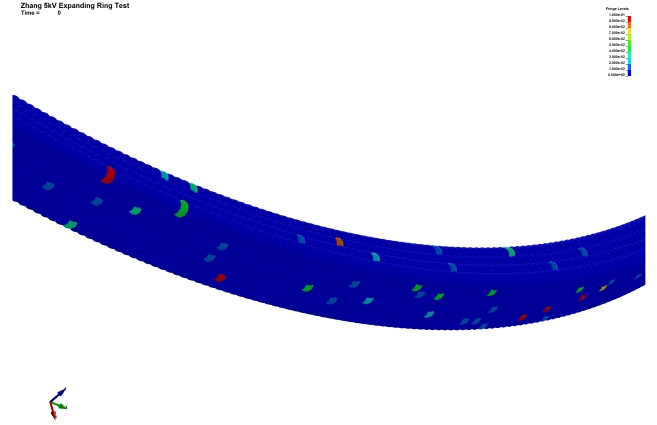
with

$$-Y = \frac{\sigma_{eq}^2}{2E(1-D)^2} \left(\frac{2}{3}(1+\nu) + 3(1-2\nu) \left(\frac{-p}{\sigma_{eq}} \right)^2 \right) \quad (27)$$

σ_{eq} is the equivalent stress (von Mises stress), ν is Poisson's ratio, E Young's modulus and p pressure. Damage D starts growing when a threshold effective plastic

Table 2 Parameters for Lemaitre damage model used for aluminium 6061-O

Property	Value
S	0.625MPa
t	1.0
D_c	0.5
$\bar{\epsilon}_{threshold}$	0.15
α	10.0
β	0.25

**Fig. 2** Initial Damage in Expanding Ring Model

strain is reached (see Equation 26), and continues until a critical damage level D_c is reached. At that point fracture occurs and the deviatoric stresses for the SPH particle under consideration are set to zero. With the evolution of damage the yield stress σ_Y and shear modulus G are degraded as follows:

$$\tilde{\sigma}_Y = \frac{\sigma_Y}{1-D} \quad (28)$$

$$\tilde{G} = \frac{G}{1-D} \quad (29)$$

The parameters used in this work are summarised in Table 2. When failure occurs the stress in the particle is set to zero and is removed from the neighbour lists thus ensuring that the particle no longer interacts directly with neighbour particles. The initial level of damage was randomised using a Weibull distribution with a mean value of 0.0024 and standard deviation of 0.00397 for the initial damage (see Figure 2). A similar approach was recently used by Meyer et al. (2012) and Hopson et al. (2011) to model statistical variations in fracture properties for ballistic impact and explosively loaded cylinders problems respectively. Since it can be assumed that the rings used in the electromagnetically driven ring experiments were free of macroscopic defects, the level of initial damage should be small. Since

the critical level of damage was 0.5 a mean initial damage level of 0.0024 appears reasonable. The results are sensitive to the Weibull distribution parameters, which is in agreement with the observations of Meyer et al. (2012) and Hopson et al. (2011). Other values for the Weibull distribution parameters would either result in too few - or too many fragments, or in fracture at a time different from the experiments. The lower expansion speed models were more sensitive to these changes in the initial damage distribution. The damage distribution is also dependent on the discretisation density. For a different discretisation density an equivalent damage distribution can be calculated using the approach described Meyer et al. (2012) and Hopson et al. (2011).

4.3 Treatment of Failure and Fracture

To allow fractures to develop realistically in SPH it is not sufficient to simply set the stress of a failed particle to zero and to remove it from the neighbour lists. As shown in Figure 3 if we consider a particle i which has a neighbour particle j_1 which has failed, then other neighbour particles, like particle j_2 , which are obscured by particle j_1 should also be removed from the neighbour list of particle i . If this is not done the stress states at particle i are unphysical. If an Eulerian kernel is used then the interaction between the particles i and j_2 which are separated by a crack would gradually reduce as they move apart. Although not physically correct a free surface would form when the particles move out of the kernel's support (ie. stop interacting). However, if a Total Lagrangian SPH formulation is used then the neighbour lists are fixed and kernels and kernel derivatives are evaluated for the initial particle locations only. Therefore it is necessary to explicitly deal with particles that are obscured by a crack (failed particles). In this study, a particle visibility was considered by introducing a truncated cone as shown in Figure 4. The top of the truncated cone is located on the failed particle. The diameter of top of the truncated cone is equal to the initial particle spacing. The angle of the cone is such that the tip of the equivalent cone is located at the i particle (see Figure 4). If a particle lies inside this cone it removed from the neighbour list of the particle i .

4.4 Electromagnetic Forces

In the SPH model, the electromagnetic loading was applied as a body force defined by the 1D analytical expression for the ring acceleration as a function of ring current proposed by Zhang and Ravi-Chandar (Zhang

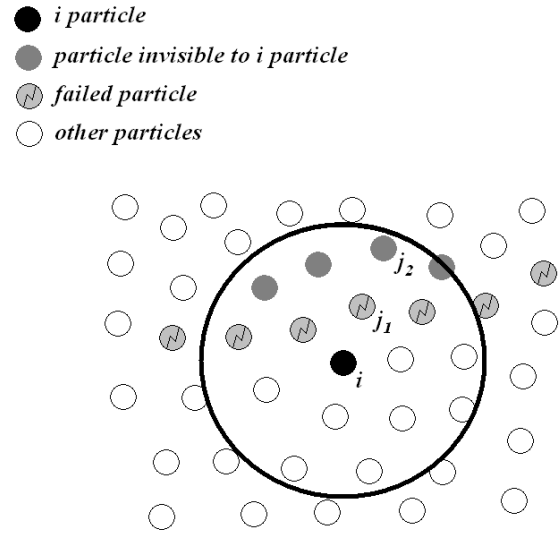


Fig. 3 Different types of neighbour particles in the presence of failed particles

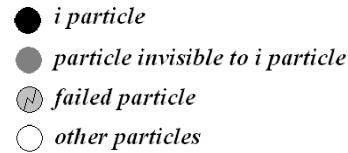


Fig. 4 Visibility criterion based on truncated cone (shaded area indicates truncated cone)

and Ravi-Chandar, 2006). The expression they used is given in Equation 30.

$$\ddot{r} = \frac{I_1 I_2}{2\pi r A \rho} \frac{dL_{12}}{dr} + \frac{I_2^2}{4\pi r A \rho} \frac{dL_{22}}{dr} - \frac{\sigma}{\rho r} \quad (30)$$

This means that the loading has been applied by using the measured currents in the ring and solenoid as input data. The effect of the resulting electromagnetic forces was incorporated by adding a body force equivalent to the first two terms in Equation 30 to the right hand

side of the momentum Equation 18. The terms L_{22} and L_{21} are function of the radius as defined in Equations 31 and 33 and hence need to be recalculated at every timestep.

$$L_{22} \approx 4\pi r \left[\ln \frac{8r}{r_w} - \frac{7}{4} \right] 10^{-7} \quad (31)$$

where

$$r_w = \sqrt{\frac{A}{\pi}} \quad (32)$$

$$\frac{1}{2\pi r} \frac{dL_{12}}{dr} = \frac{\mu_0}{2\pi} \sum_{i=0}^n \int_0^\pi \frac{r_i^2 - r_i r \cos \theta}{(r^2 - 2rr_i \cos \theta + r_i^2 + z_i^2)^{\frac{3}{2}}} d\theta \quad (33)$$

Since these terms depend on the radius of the ring it is clear that the electro-magnetical and thermo-mechanical models are coupled.

4.5 Effect of Joule Heating

The effect of large currents on temperature can not be ignored. Zhang and Ravi-Chandar (2006) calculated the temperature increase, and obtain temperature increases ranging from $250^\circ C$ to almost $500^\circ C$. This increase in temperature will have a significant effect on the constitutive behaviour of the material. A similar observation has also been made by Meyers (1994). The rate of increase of temperature as a function of the ring current and plastic work are defined as follows in Zhang and Ravi-Chandar (2006).

$$\dot{T} = \frac{R_2 I_2^2}{2\pi r A C_p \rho} + \frac{\sigma \dot{\epsilon}_p}{C_p \rho} \quad (34)$$

Only the first term in Equation 34 related to the Joule heating was added to the temperature calculation for each particle. Including the temperature increase due to Joule heating in this way implies that it was assumed that the current density is homogeneous in the cross section and the cross sectional area is independent of angular position. This last assumption is only valid up to the point at which necking occurs. In the necked region the current density would increase and this would lead to higher temperatures in the neck. However, accurately dealing with this effect would require fully coupled 3D electromagnetic and thermomechanical models. Due to the uncertainty in constitutive data, the data is taken from the literature, and the complexity of modelling this effect it was not taken into account in the models.

The second term in equation 34 related to temperature increase due to plastic work is accounted for in the constitutive equation of the SPH solver using the following equation:

$$\dot{T} = \frac{\beta \sigma : \dot{\epsilon}_p}{C_p \rho} \quad (35)$$

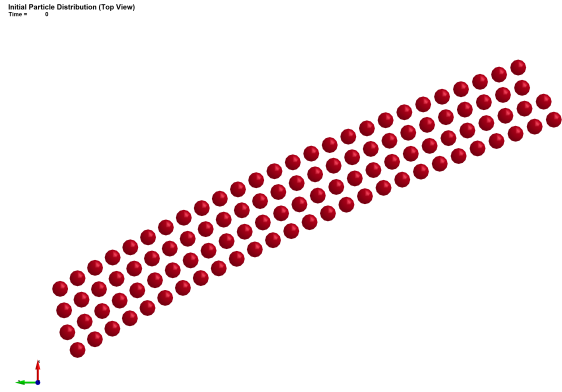


Fig. 5 Initial Positioning of Particles

This equation is the multi-axial equivalent of the second term in equation 34 and also includes the Taylor-Quinney coefficient β which defines the fraction of plastic work which is converted into heat. A value for β of 0.9 has been used in all simulations.

5 Description of Models

5.1 Particle Positioning

The initial configuration of the model is shown in Figure 2. As explained above the input loading was defined in terms of the currents measured in the primary circuit (capacitor-switch-coil) and the current measured in the ring. Although fracture occurs between $30\mu s$ and $70\mu s$, the simulations were run until $120\mu s$. This was done in order to allow for completion of the necking processes and elastic unloading of all the fragments. To avoid any preferential fracture position or direction the particles in the ring were distributed in a particular manner. The particles were positioned on a series of planes perpendicular to the axis of the ring. The distance between these planes was the integer value of the ring height divided by the desired particle spacing. Within each of these planes the particles were located on concentric circles as shown in Figure 5. The radial distance between these circles was again equal to the integer value of the ring thickness divided by desired particle spacing. The individual particles were then distributed along these circles (See Figure 5), again the number of particles on each circle was the integer value of the circle length divided by the desired particle spacing. The position of the first particle on each ring was at a random point on the circle to avoid preferential cross sectional planes along which a crack can form. A second advantage of this approach is that all particles have the same mass.

5.2 Discretisation Density

In order to determine which discretisation is able to capture localisation a sensitivity study has been performed on a bar subjected to a constant strain rate. This setup introduces similar loading to an expanding ring experiment. A similar setup was used in the Finite Element models in Zhang and Ravi-Chandar (2006) and Zhang and Ravi-Chandar (2008). The constant strain rate was achieved by fixing one end of the bar, defining a prescribed velocity v_{end} to the other end and initialising the velocity field as linearly varying from 0 to v_{end} . The bar had the same cross section as the rings (0.5 by 1.0mm) and length of 1.9164mm. In order to study localisation a defect was introduced halfway along the bar. Similarly to Zhang and Ravi-Chandar (2006) the defect was defined in the form of an area (0.250 by 0.250mm) which does not exhibit any strain hardening. This defect will cause a neck to form. The neck then develops into a complete fracture of the bar and release waves unload the rest of the bar. Three different particle spacings 0.0125mm, 0.00625mm and 0.003125mm were used, resulting in models of respectively 5664, 44800 and 365864 particles. The results are shown in figures 6 and 7. Figure 6 shows that shape of the fractured necks for the three particle spacings is similar. The evolution of the effective plastic strain for two points is shown in Figure 7. One point is located in the neck while the other point is located outside the neck. It can be seen that inside the neck the plastic strain increases rapidly during necking. The plastic strain in the point outside the neck increases linearly until that point is unloaded. This unloading occurs at a time of 25.0μs for the three particle spacings, and the plastic strain is around 0.22, the Considère strain for this material. Based on these results it was concluded that having 4 particles in the radial and 8 in the axial direction was sufficient to capture localisation. The total number of particles in the expanding ring models was 24904.

5.3 Influence of Modelling Assumptions

In order to assess the influence of the various modelling assumptions six hierarchically different models were developed and used for simulation of a range of the expanding ring experiments. The simplest model was Model 1 and the most complex model was Model 6. The results for the lowest (5kV charge) and highest (8kV charge) expansion speeds are summarised in Table 3. In the baseline Model 1 the standard Johnson-Cook model (i.e. model without the additional strain rate dependent term 24) was used and damage and the effect of plastic work on temperature were neglected. The effect



Fig. 6 Necks in bar subjected to constant strain rate

of Joule heating was taken into account. As expected in the absence of damage in the model no fragmentation occurred, however a number of necks do develop during the ring expansion. The damage model was included in Model 2. However, the initial damage was not randomised using a weibull distribution but set to 0 for all particles. Although fragmentation occurred, the number of fragments was significantly overestimated and the time at which the first fracture occurred was underestimated. The modification to the strain rate term in the constitutive model was added in Model 3. This model underestimated the number of fragments and the error in the time to the first fracture increased. The effect of plastic work on temperature was included in Model 4. This had a small influence on the fracture time. Model 5 was generated by randomising the initial damage in Model 2. In this case fragment numbers for the higher expansion speed drop significantly. Similarly the time to fracture was more than halved. Model 6 combined the previous modifications and predicts the number of fragments and the time at which the first fracture occurs quite well. This demonstrates that all relevant physics needs to be taken into account to obtain realistic results over a range of test conditions. The importance of accurate constitutive and damage models is also identified by Guduru and Freund (2002) and Zhang and Ravi-Chandar (2009b).

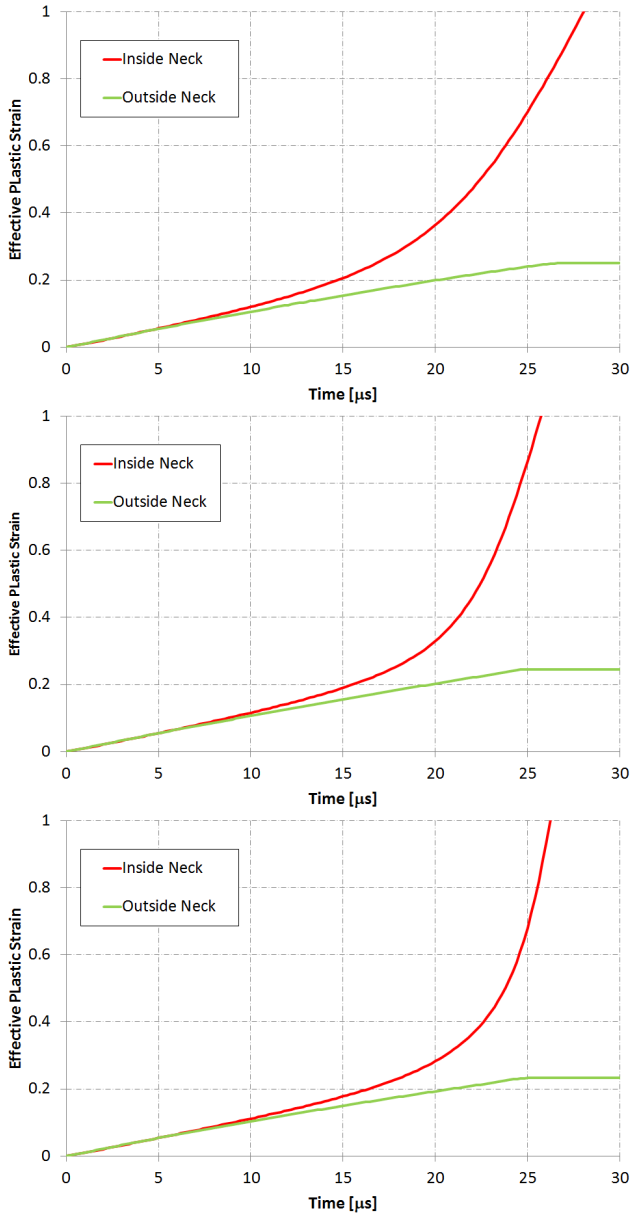


Fig. 7 Evolution of effective plastic strain in and outside the necked region in bars

6 Results and Discussion

Using these model parameters the four experiments (5kV, 6kV, 7kV and 8kV charge) were simulated. The final state (after $120\mu s$) of these four simulations is shown in Figures 8 to 15. These figures show the level of damage in the rings on a scale of 0.0 (no damage) to 0.5 (= critical damage D_c i.e. failure).

Table 3 Effect of Model Assumptions on Fragmentation Behaviour

Assumptions	5kV Experiment		8kV Experiment	
	Number of Fragments	Time of 1 st Fracture	Number of Fragments	Time of 1 st Fracture
Model 1 (Baseline Model): No damage, No strain rate modification, No heating due to plastic work, No randomisation	0	N/A	0	N/A
Model 2: No strain rate modification, No heating due to plastic work, No randomisation	10	94 μs	26	78 μs
Model 3: No heating due to plastic work, No randomisation	0	N/A	3	106 μs
Model 4: No randomisation	0	N/A	3	112 μs
Model 5: No strain rate modification, No heating due to plastic work	12	38 μs	14	30 μs
Model 6 (Final Model): with damage, strain rate modification, heating due to plastic work and randomisation	4	58 μs	12	42 μs

6.1 Number of Fragments and Necking

The number of fragments in simulations and experiments is summarised in Table 4 and Figure 16 for all tests. For the 5kV experiment it can be seen in Figures 8 and 9 that the simulation predicts 4 fragments, which compares well with the 3 fragments observed in the test. On top of complete fractures a number of arrested necks can be identified. Some necks are well developed and clearly show a reduction in cross section. Other necks are less developed but can be identified by a localised increase in damage. The width of the necks is roughly equal to the ring height which is what was observed in the experiments (Zhang and Ravi-Chandar, 2006). The simulation results for the 6kV experiment (Figure 10 and 11) are similar. In accordance with experiments the number of fragments increases. The simulation predicts 5 fragments while the 2 tests performed resulted in 6 and 8 fragments respectively. A number of arrested necks can be observed. The simulation of the 7kV ex-

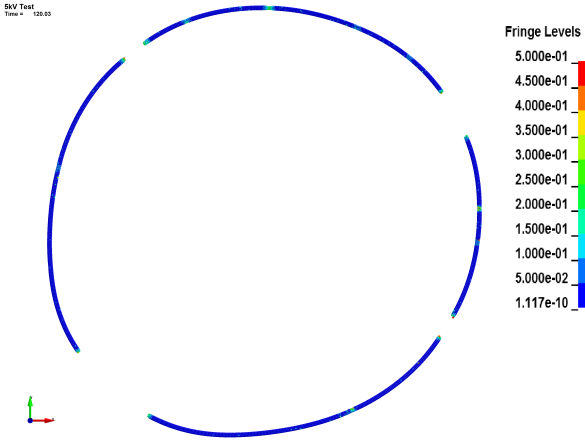


Fig. 8 Fragments and Damage Distribution in 5kv model after $120\mu s$

periment (Figure 12 and 13) results in 8 fragments (10 fragments in the test). Apart from arrested necks, in Figure 12 one can also observe a number of fragments which show some bending around the necked area. This is consistent with the test results in (Zhang and Ravi-Chandar, 2006). Finally, the simulation results for the 8kV experiment (Figure 14 and 15) result in 12 fragments (12 fragments in the test). Again arrested necks and fragments which bend around a necked area can be observed in Figure 14. It can be seen that as the expansion speed increases the number of fragments increases. In Figure 16 the number of fragments in the simulations and experiments is plot as a function of radial velocity of the ring at a time $t = 40\mu s$. Since the radial velocity is dependent on time a choice has to be made on a nominal expansion speed for each experiment. In a first phase the velocity will increase due to the electromagnetic force. As the currents in the ring drop the driving force will gradually drop to zero and from then onwards the ring will expand freely. During this stage the velocity drops as kinetic energy is converted into strain energy. The nominal expanding velocity in the tests is the velocity at the start of free expansion stage, which occurs after roughly $40\mu s$. Hence, the radial velocity in the simulations has been taken as the expansion velocity of the ring at $40\mu s$. It can be seen that the simulation results follow the same trend as the experimental data. Taking into account the scatter in experimental data, the complexity of the model and the fact many constitutive and damage parameters are taken from the literature the correlation with the experimental data is good. The RMS error between simulation and test results for the number of fragments is 1.8 fragments.

In order to check the plastic strain in the fragments the evolution of plastic strain at a number of points was plot as a function of time, similarly to section 5.2.

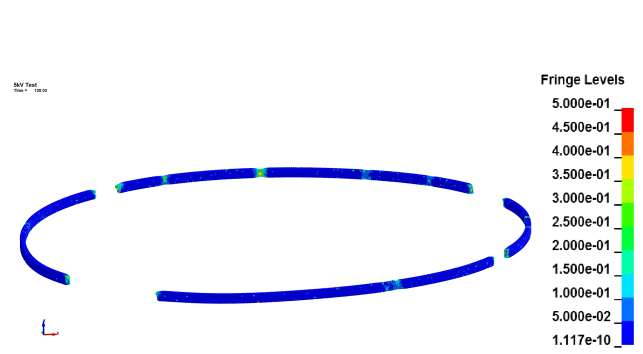


Fig. 9 Arrested Necks in 5kv model after $120\mu s$

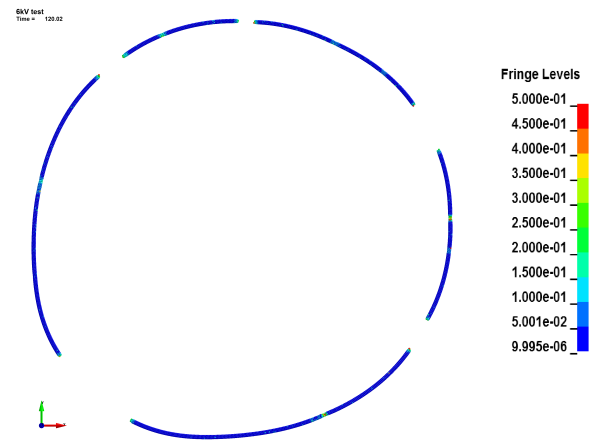


Fig. 10 Fragments and Damage Distribution in 6kv model after $120\mu s$

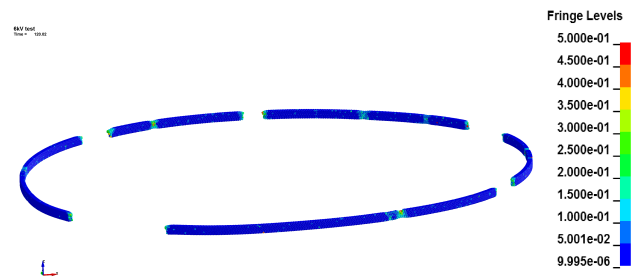


Fig. 11 Arrested Necks in 6kv model after $120\mu s$

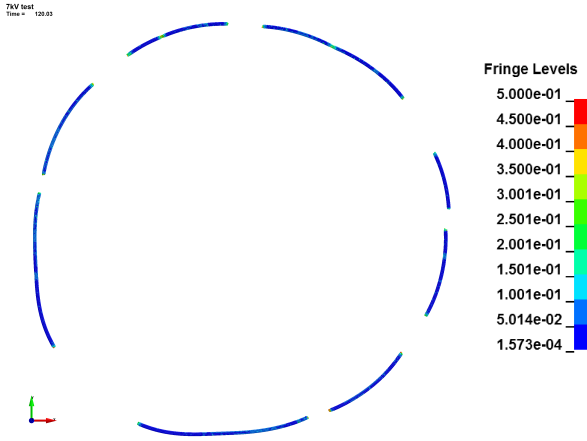


Fig. 12 Fragments and Damage Distribution in 7kV model after $120\mu s$



Fig. 13 Arrested Necks in 7kV model after $120\mu s$

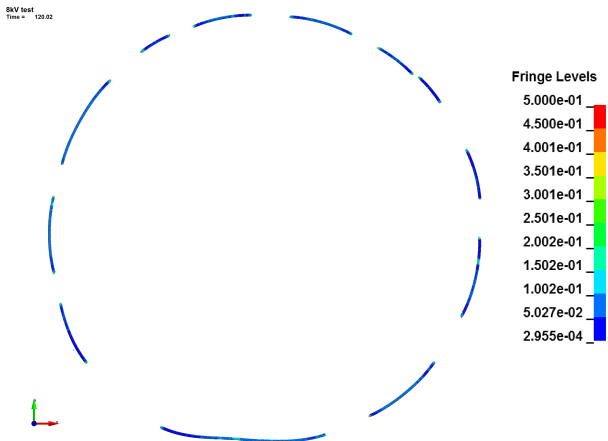


Fig. 14 Fragments and Damage Distribution in 8kV model after $120\mu s$

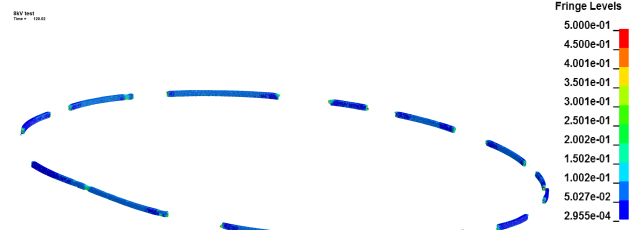


Fig. 15 Arrested Necks in 8kV model after $120\mu s$

Table 4 Comparison of Measured and Simulated Number of Fragments

Test	Number of Fragments	
	Experiment	Simulation
5kV	3	4
6kV	7	5
7kV	10	8
8kV	12	12

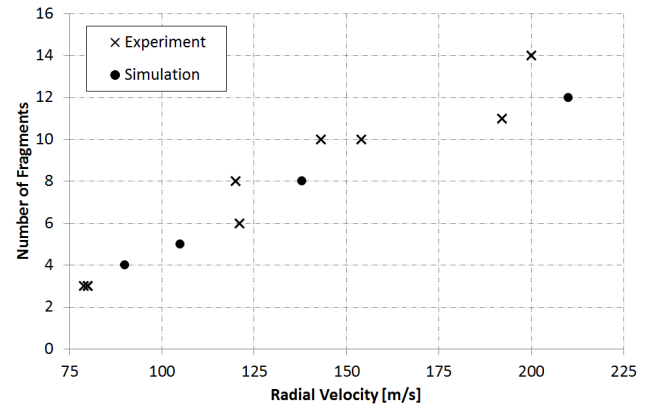


Fig. 16 Comparison of Measured and Simulated Number of Fragments as function of Radial Expansion Velocity

The results for the 5kV and 8kV tests are shown in figures 17 and 18. In both cases three points have been plot: a point in a neck, a point near the neck which first develops into a fracture, and a point diametrically opposite to this neck. It can be seen that in the neck the effective plastic strain increases rapidly during necking (red curve) and that near the first neck the strain is limited to a value close to the Considère strain (green curve). However fragments further away from the first fracture tend to show a higher strain (blue curve in the graphs), especially in the experiment at higher expan-

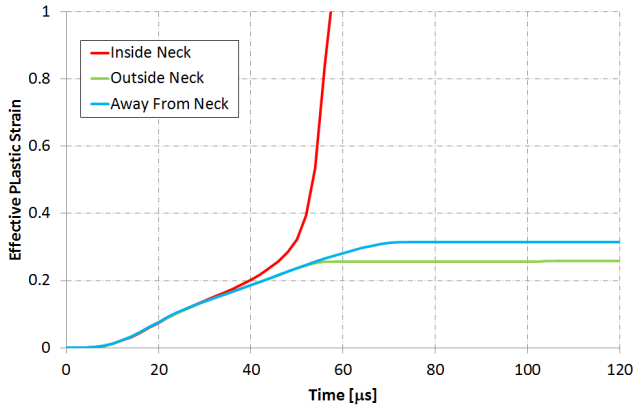


Fig. 17 Evolution of effective plastic strain versus time in and outside the necked region for the 5kV experiment

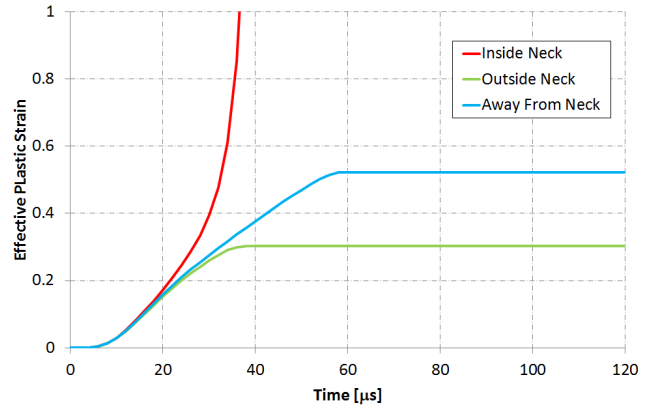


Fig. 18 Evolution of effective plastic strain versus time in and outside the necked region for the 8kV experiment

sion speed (figure 18). This can be explained by the fact that points further away from a fracture will continue to accumulate plastic deformation until a Mott release wave unloads them. This would then indicate that the development of local defects into a neck and subsequent fracture across the whole specimen cross section does not happen fast enough throughout the whole ring. In order to explain this difference it is useful summarise the findings of Zhang and Ravi-Chandar (2008). In this paper these authors discuss the effect of absolute cross section size on necking and fragmentation. They consider that localisation of deformation occurs in a small volume and that this localisation then has to propagate across the whole cross section to develop into a neck. In expanding ring tests of Al 6061-O specimen with larger cross sections they observe that the onset of necking occurs at strains of 0.45, in other words higher than the Considère strain of 0.22. They confirm this with finite element model results of samples with different cross section size. Therefore they conclude that localisation strain depends on the absolute size of the specimen and that this is due inertial effects on the propagation of the localisation across the specimen width. Based on this, a possible explanation for the higher plastic deformations away from the first necks may be that although localisation starts at the right time some localisations do not develop into necks fast enough. This may be due to inaccuracies in the constitutive and damage model and/or the randomisation of initial damage. Since the constitutive model has been used with data obtained from the literature it would be necessary to characterise the material for the range of strains, strain-rates and temperatures experienced during the tests before investigating this further. This characterisation should cover elasto-plastic behaviour as well as damage evolution.

Table 5 Comparison of Measured and Simulated Time to First Fracture

Test	Time of First Fracture	
	<i>Experiment</i>	<i>Simulation</i>
5kV	57.7 μ s	58.0 μ s
6kV	47.99 μ s	56.0 μ s
7kV	38.7 μ s	44.0 μ s
8kV	40.55 μ s	42.0 μ s

6.2 Time of First Fracture

The time at which the first fracture occurs in simulations and experiments is summarised in table 5 and Figure 19 for all tests. For the 5kV experiment it can be seen that the simulation predicts that after 57.7 μ s the first fracture occurs. This compares well to the test result of 58.0 μ s. The simulation results for the 6kV experiment shows that compared to the 5kV the time to first fracture in the simulation is reduced to 56.0 μ s, in the test this was 47.99 μ s. The simulation of the 7kV experiment results in a further reduction in time to first fracture to 44.0 μ s (38.7 μ s in the test). Finally, in the simulation results for the 8kV experiment a time to first fracture to 42.0 μ s is obtained (40.55 μ s in the test). It can be seen that as the expansion speed increases the time at which the first fracture occurs decreases. The simulation results follow the same trend as the experimental data. The correlation with the experimental data is good and the RMS error between simulation and test results for the time at which the first fracture occurs is 4.9 μ s.

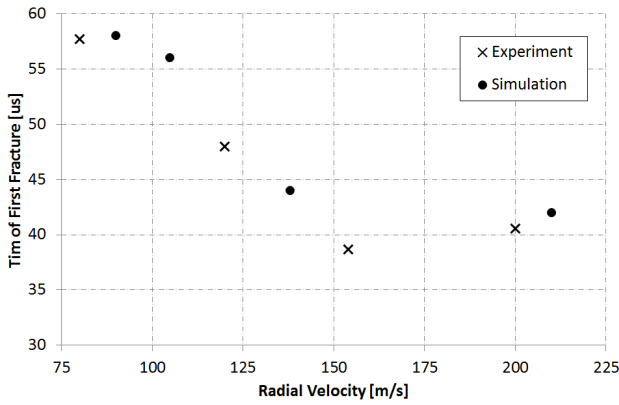


Fig. 19 Comparison of Measured and Simulated Time of First Fracture as function of Radial Expansion Velocity

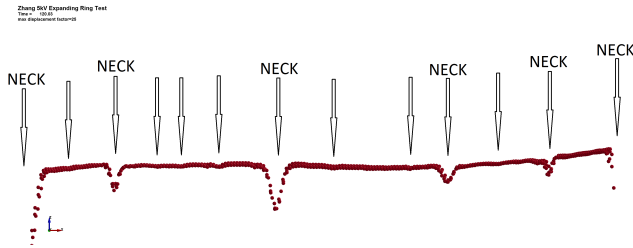


Fig. 20 Top surface of a fragment, axial displacement amplification factor = 25

6.3 Number of Necks

As mentioned in section 6.1 during the simulation a number of necks develop. Some of these necks develop into fractures. Other necks are arrested and bending of some fragments around an arrested neck can be observed. These observations are in agreement with the experimental data. For completeness this section contains the comparison of experimentally measured and simulated number of necks. The experimental results in Zhang and Ravi-Chandar (2006) are based on measuring the variation of thickness of the ring. Hence a similar approach was used for the determination of necks in the simulation results. However, since in the experiments no criterion was provided to determine when a variation in thickness is a neck, only clearly observable necks in the simulation results have been included. To illustrate this a typical top surface obtained in the simulations is shown in figure 20. In order to make it easier to identify thickness variations the displacement in the axial direction has been amplified by a factor of 25. On this figure variations of thickness are indicated with a arrow and labelled when they were considered a neck. The number of necks in simulations and experiments is summarised in table 6 and Figure 21 for all tests. The number of clearly identifiable necks is

Table 6 Comparison of Measured and Simulated Number of Necks

Test	Number of Necks	
	Experiment	Simulation
5kV	17	14
6kV	20	16
7kV	29	20
8kV	32	25

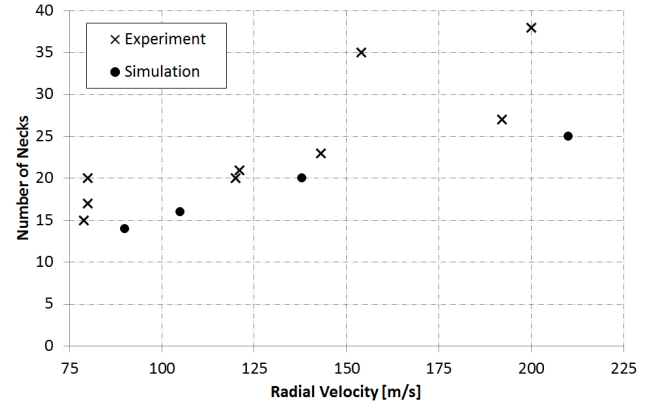


Fig. 21 Comparison of Measured and Simulated Number of Necks

lower than the number of necks reported in Zhang and Ravi-Chandar (2006) but the overall trend of increasing number of necks with increasing expansion speed is well captured. The RMS error is 7.4. The lower number of identified necks in the simulations may be due several factors. Firstly, it may be due to differences in the criterion used to consider a thickness variation as a neck. Secondly, the number of necks depends on hardening sensitivities. The Johnson-Cook model was used in this work because input data was available. Therefore the use of constitutive laws which are more accurate at describing hardening over a range of strain rates and temperatures, for example Rusinek-Klepaczko (Rusinek and Zaera, 2007) or Preston-Tonks-Wallace (Mercier et al., 2010) may provide better agreement with experimental results. In the work by Mercier et al. (2010) it was in fact shown that for pure copper and tantalum the Preston-Tonks-Wallace constitutive equation predicts more necks (as observed in experiments) than simpler constitutive laws such as Zerilli-Armstrong and a power law.

7 Conclusion

The modelling of necking and fracture at high strain rates, and in particular modelling of electromagneti-

cally driven ring experiments has been studied in this paper. It has been demonstrated that a total Lagrangian Smoothed Particle Hydrodynamics method with normalised kernel is a practical discretisation technique for this problem. However, in order to correctly model fracture in this total Lagrangian SPH formulation it was necessary to implement a visibility criterion based on a truncated cone which removes particles from the neighbour list if they are obscured by a failed particle. This method was implemented in 3D and has been demonstrated to be effective and robust. The material behaviour has been modelled with a Johnson-Cook plasticity model in combination with a Lemaitre damage model. The effect of Joule heating due to the current induced in the ring was taken into account in the constitutive model. The electromagnetic and mechanical models are coupled since the electromagnetic force is dependent on the radius of the ring, and the electromagnetic force drives the ring and the constitutive equation is function of temperature which is affected by the Joule heating in the ring. Therefore the loading has been applied by using the measured currents in the ring and by solving a 1D electromagnetic model and the 3D SPH thermo-mechanical model. Using this modelling approach the number of fragments was predicted with an RMS error of 1.8 fragments and the time at which the first fracture occurs was predicted with an RMS error of $4.9\mu s$. In agreement with experimental observations the simulations also predicted the presence of arrested necks and fragments exhibiting bending around an arrested neck. The trend of increase in number of necks with increasing expansion speed is captured, however the number of necks is underestimated in the simulation results with the RMS error of 7.4.

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Total Lagrangian SPH modelling of necking and fracture in electromagnetically driven rings

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2013-03-31T00:00:00Z

Springer Science Business Media

<http://dx.doi.org/10.1007/s10704-012-9801-4>

T. De Vuyst, R. Vignjevic, Total Lagrangian SPH modelling of necking and fracture in electromagnetically driven rings, International Journal of Fracture, March 2013, Volume 180, Issue 1, Pages 53-70.